

ASME B1.30-2002
(Revision of ASME B1.30M-1992)

SCREW THREADS: STANDARD PRACTICE FOR CALCULATING AND ROUNDING DIMENSIONS

AN AMERICAN NATIONAL STANDARD

Intentionally left blank

ASMENORMDOC.COM : Click to view the full PDF of ASME B1.30 2002



The American Society of
Mechanical Engineers

A N A M E R I C A N N A T I O N A L S T A N D A R D

SCREW THREADS: STANDARD PRACTICE FOR CALCULATING AND ROUNDING DIMENSIONS

ASME B1.30-2002
ASMENORMDOC.COM : Click to view the full PDF of ASME B1.30-2002

ASME B1.30-2002
Revision of ASME B1.30M-1992

Date of Issuance: April 25, 2003

This Standard will be revised when the Society approves the issuance of a new edition. There will be no addenda or written interpretations of the requirements of this Standard issued to this edition.

ASME is the registered trademark of The American Society of Mechanical Engineers.

This code or standard was developed under procedures accredited as meeting the criteria for American National Standards. The Standards Committee that approved the code or standard was balanced to assure that individuals from competent and concerned interests have had an opportunity to participate. The proposed code or standard was made available for public review and comment that provides an opportunity for additional public input from industry, academia, regulatory agencies, and the public-at-large.

ASME does not "approve," "rate," or "endorse" any item, construction, proprietary device, or activity.

ASME does not take any position with respect to the validity of any patent rights asserted in connection with any items mentioned in this document, and does not undertake to insure anyone utilizing a standard against liability for infringement of any applicable letters patent, nor assume any such liability. Users of a code or standard are expressly advised that determination of the validity of any such patent rights, and the risk of infringement of such rights, is entirely their own responsibility.

Participation by federal agency representative(s) or person(s) affiliated with industry is not to be interpreted as government or industry endorsement of this code or standard.

ASME accepts responsibility for only those interpretations of this document issued in accordance with the established ASME procedures and policies, which precludes the issuance of interpretations by individuals.

No part of this document may be reproduced in any form,
in an electronic retrieval system or otherwise,
without the prior written permission of the publisher.

The American Society of Mechanical Engineers
Three Park Avenue, New York, NY 10016-5990

Copyright © 2003 by
THE AMERICAN SOCIETY OF MECHANICAL ENGINEERS
All rights reserved
Printed in U.S.A.

CONTENTS

Foreword	iv
Committee Roster	v
1 General	1
2 Rounding of Decimal Values	1
3 Calculations From Formulas	2
4 Thread Form Constants	2
Tables	
1 Number of Decimal Places Used in Calculations	2
2 Example of External Inch Screw Threads, $\frac{1}{2}$ -28 UNEF-2A	4
3 Example of Internal Inch Screw Threads, $\frac{1}{2}$ -28 UNEF-2B	7
4 Example of External Inch Screw Threads, $\frac{19}{64}$ -36 UNS-2A	9
5 Example of Internal Inch Screw Threads, $\frac{19}{64}$ -36 UNS-2B	12
6 Example of External Metric Screw Threads, M12 $\times$ 1.75-4g6g	14
7 Example of Internal Metric Screw Threads, M12 $\times$ 1.75-6H	16
8 Example of External Metric Screw Threads, M13 $\times$ 0.9-4g6g	18
9 Example of Internal Metric Screw Threads, M13 $\times$ 0.9-6H	20
10 Thread Form Data	22
Nonmandatory Appendix	
A Thread Characteristics Governed by this Standard	23

FOREWORD

The American National Standards Committee B1 for standardization of screw threads was organized in 1920 as sectional Committee B1 under the sponsorship of the American Engineering Standards Committee (later the American Standards Association, then the United States of America Standards Institute and as of October 6, 1969, the American National Standards Institute, Inc.), with the Society of Automotive Engineers and The American Society of Mechanical Engineers as joint sponsors. As a result, a great deal of effort was expended through the years toward development of several screw thread standards.

Recognizing the need to standardize the method of rounding decimal values in the calculation of screw thread dimensions, the ASME Standards Committee B1 charged Subcommittee 30 with the responsibility for producing such a standard.

This Standard designated ASME B1.30 was developed only to serve as the basis for rounding of decimal values associated with the computation of screw thread dimensions.

The examples and formulas depicted within this Standard are for reference only and are presented only to clarify the rounding procedures described. When calculating the different thread characteristics for a particular thread form, refer to the appropriate ASME B1 standard for the formulas and methodology of calculation for that thread.

This Standard was originally approved as an American National Standard on June 29, 1992. This issue updates the 1992 issue and includes:

- (a) the addition of a note to para. 1.2 for the clarification of the use of this Standard for metric applications;
- (b) the addition of an exception in para. 3.1.3 to the number of decimal places for the allowance 2A when used in intermediate calculations as Td_2 ;
- (c) the addition of the maximum external UN minor diameter, maximum and minimum internal pitch diameters and the minimum internal major diameter to the example in para. 3.2.1;
- (d) removal of Appendix A containing various common values for 60 deg thread elements. This was done because the values are either listed or can be calculated by formulas in other existing B1 standards;
- (e) the addition of an example for a standard metric size that is listed in ASME B1.13M and ISO 261;
- (f) the addition of an example for a special inch size showing how to round numbers with an infinite number of digits after the decimal point;
- (g) added a new Appendix A with a table listing the number of decimal places for each of the thread characteristics used in the examples in B1.30.

Suggestions for improvement of this Standard will be welcome. They should be sent to Secretary, ASME B1 Standards Committee, Three Park Avenue, New York, NY 10016-5990 U.S.A.

This revision was approved as an American National Standard on July 12, 2002.

ASME STANDARDS COMMITTEE B1

Standardization and Unification of Screw Threads

(The following is the roster of the Committee at the time of approval of this Standard.)

OFFICERS

A. L. Barrows, *Chair*
D. S. George, *Vice Chair*
J. R. Bird, *Secretary*

COMMITTEE PERSONNEL

G. L. Allen, The L. S. Starrett Co.
A. L. Barrows, Kennametal-IPG
M. H. Bernier, Standard Nut and Bolt
J. R. Bird, The American Society of Mechanical Engineers
F. G. Calderone, Quality Systems Implementers
J. Davis, U.S. Nuclear Regulatory Commission
L. N. Dixon, Jr., General Electric
R. Dodge, Pennoyer-Dodge Co.
G. A. Flannery, Mercury Gage Co.
D. Skierski, *Alternate*, Mercury Gage Co.
H. N. Frost, Defense Supply Center Philadelphia
D. S. George, Ford Motor Co.
J. R. Gervasi, Kerr Lakeside Inc.
J. Greenslade, Greenslade & Co.
L. C. Johnson, The Johnson Gage Co.
D. D. Katz, Precision Fittings
R. P. Knittel, Leitech / AMTMA
B. Larzelere, Deltronic Corp.

L. L. Lord, Caterpillar, Inc.
R. L. Tennis, *Alternate*, Caterpillar, Inc.
M. H. McWilliams, PMC Gage, Inc.
D. Miskinis, Kennametal-IPG
W. R. Newman, Facil LLC, USA
M. W. Rose, Southern Gage, Inc.
D. M. Satava, Consultant
E. Schwartz, Consultant
R. H. Searr, Conam Quantum Inspection and Testing
R. E. Seppely, Consultant
A. D. Shepherd, Jr., Emuge Corp.
A. G. Strang, Consultant
R. D. Strong, General Motors Corp.
A. F. Thibodeau, Swanson Tool Manufacturing, Inc.
R. C. Veale, Consultant
R. E. Vincent, Jr., General Plug Manufacturing Co.
C. J. Wilson, Industrial Fasteners Institute
F. W. Akstens, *Alternate*, Industrial Fasteners Institute

SUBCOMMITTEE 30 — ROUNDING OF VALUES

R. H. Searr, *Chair*, Conam Quantum Inspection and Testing
A. L. Barrows, *Vice Chair*, Kennametal-IPG
B. I. Armstrong, Naval Warfare Assessment Division
R. Dodge, Pennoyer-Dodge Co.
D. S. George, Ford Motor Co.
J. Greenslade, Greenslade & Co.
L. C. Johnson, The Johnson Gage Co.
D. D. Katz, Precision Fittings
W. R. Newman, Facil LLC, USA

M. W. Rose, Southern Gage, Inc.
D. M. Satava, Consultant
E. Schwartz, Consultant
R. E. Seppely, Consultant
A. D. Shepherd, Jr., Emuge Corp.
A. G. Strang, Consultant
R. D. Strong, General Motors Co.
R. C. Veale, Consultant
C. J. Wilson, Industrial Fasteners Institute
F. W. Akstens, *Alternate*, Industrial Fasteners Institute

Intentionally left blank

ASMENORMDOC.COM : Click to view the full PDF of ASME B1.30 2002

SCREW THREADS: STANDARD PRACTICE FOR CALCULATING AND ROUNDING DIMENSIONS

1 GENERAL

1.1 Purpose

The purpose of this Standard is to establish uniform and specific practices for calculating and rounding the numeric values used for inch and metric screw thread design data dimensions only. No attempt is made to establish a policy of rounding actual thread characteristics measured by the manufacturer or user of thread gages. Covered is the Standard Rounding Policy regarding the last figure or decimal place to be retained by a numeric value and the number of decimal places to be retained by values used in intermediate calculations of thread design data dimensions. Values calculated to this Standard for inch and metric screw thread design data dimensions may vary slightly from values shown in existing issues of ASME B1 screw thread standards and are to take precedence in all new or future revisions of ASME B1 standards as applicable except as noted in para. 1.2.

1.2 Metric Application

Allowances (fundamental deviations) and tolerances for metric M and MJ screw threads are based upon formulas, which appear in applicable standards. Values of allowances for standard tolerance positions and values of tolerances for standard tolerance grades are tabulated in these standards for a selection of pitches. Rounding rules specified below have not been applied to these values but have followed practices of the International Organization for Standardization (ISO). For pitches, which are not included in the tables, standard formulas and the rounding rules specified herein are applicable.

NOTE: ISO rounding practices, for screw thread tolerances and allowances, use rounding to the nearest values in the R40 series of numbers in accordance with ISO 3. In some cases, the rounded values have been adjusted to produce a smooth progression. Since the ISO rounded values have been standardized internationally, for metric screw threads, it would lead to confusion if tolerances and allowances were recalculated using B1.30 rules, for use in the USA. B1.30 rounding rules are, therefore, only applicable to special threads where tabulated values do not exist in ISO standards. Values calculated using the ISO R40 series values may differ from those calculated using B1.30. In such a case the special thread values generated using B1.30 take precedence.

2 ROUNDING OF DECIMAL VALUES

The following rounding practice represents the method to be used in all new or future revisions of ASME B1 thread standards.

NOTE: It is recognized that this Standard is not in agreement with other published documents, e.g., ASME SI-9, Guide for Metrication of Codes and Standards SI (Metric) Units, and IEEE/ASTM SI 10, Standard for Metric Practice. The rounding practices used in the forenamed documents are designed to produce even distribution of numerical values. The purpose of this document is to define the most practical and common used method of rounding numerical thread form values. Considering today's technology, application of this method is far more practical in the rounding of thread form values.

2.1 Rounding Policy

2.1.1 When the figure next beyond the last figure or place retained is less than 5, the figure in the last place retained is kept unchanged.

EXAMPLE:

1.012342	1.01234
1.012342	1.0123
1.012342	1.012

2.1.2 When the figure next beyond the last figure or place retained is greater than 5, the figure in the last place retained is increased by 1.

EXAMPLE:

1.56789	1.5679
1.56789	1.568
1.56789	1.57

2.1.3 When the figure next beyond the last figure or place retained is 5, and:

(a) there are no figures, or only zeros, beyond this 5, the last figure should be increased by 1;

EXAMPLE:

1.01235	1.0124
1.0123500	1.0124
1.012345	1.01235
1.01234500	1.01235

(b) if the 5 next beyond the figure in the last place to be retained are followed by any figures other than zero,

Table 1
Number of Decimal Places Used in Calculations

Units	Pitch	Constants	Intermediate	Final
Inch	8	8	6	4 (except as outlined in para. 3.1.3)
Metric	as designated	7	5	3

the figure in the last place retained should be increased by 1.

EXAMPLE:

1.0123501	1.0124
1.0123599	1.0124
1.01234501	1.01235
1.01234599	1.01235

2.1.4 The final rounded value is obtained from the most precise value available and not from a series of successive rounding. For example, 0.5499 should be rounded to 0.550, 0.55 and 0.5 (not 0.6), since the most precise value available is less than 0.55. Similarly, 0.5501 should be rounded as 0.550, 0.55 and 0.6, since the most precise value available is more than 0.55. In the case of 0.5500 rounding should be 0.550, 0.55 and 0.6, since the most precise value available is 0.5500.

3 CALCULATIONS FROM FORMULAS

3.1 General Rules

3.1.1 Values for pitch and constants derived from a function of pitch are used out to eight decimal places for inch series. The eight place values are obtained by rounding their truncated ten place values. Seven decimal place values for metric series constants are derived by rounding their truncated nine place values. See para. 4 for thread form constants. Values used in intermediate calculations are rounded to two places beyond the number of decimal places retained for the final value, see Table 1.

EXAMPLE:

$$\begin{aligned}
 n &= 28 \text{ threads per inch} \\
 P &= 1/n \\
 P &= 1/28 \\
 P &= 0.0357142857 \\
 P &= 0.03571429
 \end{aligned}$$

3.1.2 Rounding to the final value is the last step in a calculation.

3.1.3 For inch screw thread dimensions, four decimal places are required for the final values of pitch diameter, major diameter, and minor diameter with the exception of Class 1B and 2B internal thread minor diameters for thread sizes 0.138 and larger, see below. The final values for the allowances and tolerances applied to thread elements are expressed to four decimal places except for

external thread pitch diameter tolerance, Td_2 , which is expressed to six decimal places.

(a) *Minor Diameter Exceptions for Internal Threads:*

(1) *Minimum Minor Diameter.* All classes are calculated and then rounded off to the nearest 0.001 in. and expressed in three decimal places for sizes 0.138 in. and larger. For Class 3B, a zero is added to yield four decimal places.

(2) *Maximum Minor Diameter.* All classes are calculated before rounding, then rounded for Classes 1B and 2B to the nearest 0.001 in. for sizes 0.138 in. and larger. Class 3B values are rounded to four decimal places.

3.1.4 Metric screw threads are dimensioned in millimeters. The final values of pitch diameter, major diameter, minor diameter, allowance and thread element tolerances are expressed to three decimal places.

3.1.5 Values containing multiple trailing zeros out to the required number of decimal places can be expressed by displaying only two of them beyond the last significant digit.

EXAMPLE: 20 threads per inch has a pitch equal to 0.05000000 and can be expressed as 0.0500.

3.2 Examples

The formulas in the examples for inch screw threads in Tables 2, 3, 4 and 5 are based on those listed in ASME B1.1, Unified Inch Screw Threads. Tables 2 and 3 are based on a size that when converted from a fraction to a decimal will result in a number that has only four decimal places. Tables 4 and 5 are based on a size that when converted will result in a number with infinite numbers of digits after the decimal point.

The formulas in the examples for metric screw threads in Tables 6, 7, 8 and 9 are based on those listed in ASME B1.13M, Metric Screw Threads. Tables 6 and 7 show the use of the tabulated values for allowances and tolerances (in accordance with ISO 965-1) for the calculation of size limits for standard diameter/pitch combinations listed in both ISO 261 and ASME B1.13M. The constant values differ from those used for inch screw threads, in accordance with the policy of rounding of this Standard, because metric limits of size are expressed to only three decimal places rather than four.

4 THREAD FORM CONSTANTS

For thread form data tables, see Table 10. The number of decimal places and the manner in which they are

listed should be consistent. Thread form constants printed in older thread standards are based on a function of thread height (H) or pitch (P). The equivalent of the corresponding function is also listed. There are some constants that would require these values to 8 or 7 decimal places before they would round to equivalent values. For standardization the tabulated listing of thread

values based on a function of pitch has been established, with thread height used as a reference only. All thread calculations are to be performed using a function of pitch (P), rounded to 8 decimal places for inch series and as designated for metric series, not a function of thread height (H). Thread height is to be used for reference only.

ASMENORMDOC.COM : Click to view the full PDF of ASME B1.30 2002

Table 2 Example of External Inch Screw Threads, $\frac{1}{2}$ -28 UNEF-2A

Thread Size = $\frac{1}{2}$ -28 or 0.5000-28 UNEF-2A		Example of Size Calculation	Additional Information
Characteristic Description			
(1) Maximum external major diameter (d max.) = Basic major diameter (d bsc) – allowance (es)		d max. = d bsc – es d bsc = 0.5000 $es = 0.300$ (Td_2 for class 2A) $es = 0.300$ (0.003668) $es = 0.001100$ $es = 0.0011$ d max. = $0.5000 - 0.0011$ d max. = 0.4989	... This is the final value of the basic major diameter which is rounded to four decimal places. For the Class 2A pitch diameter tolerance, see Table 2 (4). The six decimal place value for Class 2A pitch diameter tolerance (Td_2) is used in this calculation. This figure is rounded to four decimal places to obtain the final value of es.
(2) Minimum external major diameter (d min.) = Maximum external major diameter (d max.) – major diameter tolerance (Td)		d min. = d max. – Td $Td = 0.060 \sqrt[3]{\frac{1}{P^2}}$ $Td = 0.060 \sqrt[3]{(0.03571429)^2}$ $Td = 0.060 \sqrt[3]{0.001276}$ $Td = 0.060$ (0.108463) $Td = 0.006508$ $Td = 0.0065$ d min. = $0.4989 - 0.0065$ d min. = 0.4924	... All thread calculations are to be performed using (P) rounded to 8 decimal places. This figure is rounded to four decimal places to obtain the final value of Td.

Table 2 Example of External Inch Screw Threads, 1/2-28 UNEF-2A (Cont'd)

Thread Size = 1/2-28 or 0.5000-28 UNEF-2A		Additional Information
Characteristic Description	Example of Size Calculation	
(3) Maximum external pitch diameter (d_{2max}) = Maximum external major diameter (d_{max}) - twice the external thread addendum ($2h_{as}$)	$d_{2max} = d_{max} - 2h_{as}$ $2h_{as} = 0.64951905P$ $2h_{as} = 0.64951905 (0.03571429)$ $2h_{as} = 0.023197$ $d_{2max} = 0.4989 - 0.023197$ $d_{2max} = 0.475703$ $d_{2max} = 0.4757$	 All thread calculations are to be performed using (P) rounded to 8 decimal places. This figure is rounded to four decimal places to obtain the final value of d_{2max}. ...
(4) Minimum external pitch diameter (d_{2min}) = Maximum external pitch diameter (d_{2max}) - external pitch diameter tolerance (Td_2)	$d_{2min} = d_{2max} - Td_2$ $Td_2 = 0.0015 \sqrt[3]{d_{BSC} + 0.0015 \sqrt{LE} + 0.015 \sqrt[3]{P^2}}$ $Td_2 = 0.0015 \sqrt[3]{0.5000 + 0.0015 \sqrt{0.03571429} + 0.015 \sqrt[3]{(0.03571429)^2}}$ $Td_2 = 0.0015 \sqrt[3]{0.5000 + 0.0015 \sqrt{0.321429} + 0.015 \sqrt[3]{0.001276}}$ $Td_2 = (0.0015) (0.793701) + (0.0015) (0.566647) + (0.015) (0.108463)$ $Td_2 = 0.001191 + 0.000850 + 0.001627$ $Td_2 = 0.003668$ $d_{2min} = 0.4757 - 0.003668$ $d_{2min} = 0.472032$ $d_{2min} = 0.4720$	 LE (length of engagement) required in this example, is equal to $9P$. See ASME B1.1 for LE applications for other thread series. This figure is rounded to six decimal places to obtain the final value of the external pitch diameter tolerance Td_2. ... This figure is rounded to four decimal places to obtain the final value of d_{2min}. ...

Table 2 Example of External Inch Screw Threads, 1/2-28 UNEF-2A (Cont'd)

Thread Size = 1/2-28 or 0.5000-28 UNEF-2A		
Characteristic Description	Example of Size Calculation	Additional Information
(5) Maximum external UNR minor diameter (d_3 max.) = Maximum external major diameter (d max.) – double height of external UNR thread ($2h_s$)	d_3 max. = d max. – $2h_s$ $2h_s = 1.19078493P$ $2h_s = 1.19078493 (0.03571429)$	 All thread calculations are to be performed using (P) rounded to 8 decimal places. This figure is rounded to four decimal places to obtain the final value of d_3 max. ...
(6) Maximum external UN minor diameter (d_1 max.) = Maximum external major diameter (d max.) – double height of external UN thread ($2h_s$)	$2h_s = 0.042528$ d_3 max. = $0.4989 - 0.042528$ d_3 max. = 0.456372 d_3 max. = 0.4564	 For UN threads, $2h_s = 2h_n$ All thread calculations are to be performed using (P) rounded to 8 decimal places. This figure is rounded to four decimal places to obtain the final value of d_1 max. ...

GENERAL NOTES:

(a) All dimensions expressed in inches for calculations in this table.

(b) $P = 1/n = 1/28 = 0.03571429$.

Table 3 Example of Internal Inch Screw Threads, 1/2-28 UNEF-2B

Thread Size = 1/2-28 or 0.5000-28 UNEF-2B		
Characteristic Description	Example of Size Calculation	Additional Information
(1) Minimum internal minor diameter (D_1 min.) = Basic major diameter (D bsc) – double height of internal thread ($2h_n$)	$D_1 \text{ min.} = D \text{ bsc} - 2h_n$ $2h_n = 1.08253175P$ $2h_n = 1.08253175 (0.03571429)$ $2h_n = 0.038662$ $D_1 \text{ min.} = 0.5000 - 0.038662$ $D_1 \text{ min.} = 0.461338$	... All thread calculations are to be performed using (P) rounded to 8 decimal places. ... For the Class 2B thread used in this example this figure is rounded to three decimal places to obtain the final value of D_1 min. Other sizes and classes are expressed in a four place decimal. See para. 3.1.3(a)(1). ...
(2) Maximum internal minor diameter (D_1 max.) = Minimum internal minor diameter (D_1 min.) rounded to six decimal places + internal minor diameter tolerance (TD_1)	$D_1 \text{ min.} = 0.461$ $D_1 \text{ max.} = D_1 \text{ min. (to six decimal places)} + TD_1$ $TD_1 = 0.25P - 0.4P^2$ $TD_1 = 0.25 (0.03571429) - 0.4 (0.03571429)^2$ $TD_1 = 0.25 (0.03571429) - 0.4 (0.001276)$ $TD_1 = 0.008929 - 0.000510$ $TD_1 = 0.008419$ $TD_1 = 0.0084$ $D_1 \text{ max.} = 0.461338 + 0.0084$ $D_1 \text{ max.} = 0.469757$	See para. 3.1.3(a)(2) for exception rule when calculating D_1 max. See ASME B1.1 for limitations on use of this formula. All thread calculations are to be performed using (P) rounded to 8 decimal places. ... This figure is rounded to four decimal places to obtain the final value of TD_1. ... For the Class 2B thread used in this example this figure is rounded to three decimal places to obtain the final value of D_1 max. Other sizes and classes are expressed in a four place decimal. See para. 3.1.3(a). ...
	$D_1 \text{ max.} = 0.470$	

Table 3 Example of Internal Inch Screw Threads, 1/2-28 UNEF-2B (Cont'd)

Thread Size = 1/2-28 or 0.5000-28 UNEF-2B		
Characteristic Description	Example of Size Calculation	Additional Information
(3) Minimum internal pitch diameter (D_2 min.) = Basic major diameter (D bsc) – twice the external thread addendum (h_b)	D_2 min. = D bsc – h_b $h_b = 0.64951905P$ $h_b = 0.64951905 (0.03571429)$ $h_b = 0.023197$ D_2 min. = $0.5000 - 0.023197$ D_2 min. = 0.476803 D_2 min. = 0.4768	 All thread calculations are to be performed using (P) rounded to 8 decimal places. This figure is rounded to four decimal places to obtain the final value of D_2 min. ...
(4) Maximum internal pitch diameter (D_2 max.) = Minimum internal pitch diameter (D_2 min.) + internal pitch diameter tolerance (TD_2)	D_2 max. = D_2 min. + TD_2 $TD_2 = 1.300 (TD_2 \text{ for class } 2A)$ $TD_2 = 1.300 (0.003668)$ $TD_2 = 0.004768$ $TD_2 = 0.0048$ D_2 max. = $0.4768 + 0.0048$ D_2 max. = 0.4816	... The constant 1.300 is for this Class 2B example and will be different for Classes 1B and 3B. See ASME B1.1. For the TD_2 Class 2A pitch diameter tolerance, see Table 2 (4). The six place decimal place value is used. This figure is rounded to four decimal places to obtain the final value of TD_2
(5) Minimum internal major diameter (D min.) = Basic major diameter (D bsc)	D min. = D bsc D bsc = 0.5000 D min. = 0.5000	

GENERAL NOTES:

(a) All dimensions expressed in inches for calculations in this table.

(b) $P = 1/n = 1/28 = 0.03571429$.

Table 4 Example of External Inch Screw Threads, $\frac{19}{64}$ -36 UNS-2A

Thread Size = $\frac{19}{64}$ -36 or 0.2969 (0.296875 Rounded to Four Decimal Places)-36 UNS-2A		Example of Size Calculation	Additional Information
Characteristic Description			
(1) Maximum external major diameter (d max.) = Basic major diameter (d bsc) – allowance (es)		$d \text{ max.} = d \text{ bsc} - es$ $d \text{ bsc} = 0.2969$ $es = 0.300$ (Td_2 for Class 2A) $es = 0.300$ (0.003127) $es = 0.000938$ $es = 0.0009$ $d \text{ max.} = 0.2969 - 0.0009$ $d \text{ max.} = 0.2960$	... This is the final value of the basic major diameter which is rounded to four decimal places. For the Class 2A pitch diameter tolerance, see Table 4 (4). The six decimal place value for Class 2A pitch diameter tolerance (Td_2) is used in this calculation. This figure is rounded to four decimal places to obtain the final value of es.
(2) Minimum external major diameter (d min.) = Maximum external major diameter (d max.) – major diameter tolerance (Td)		$d \text{ min.} = d \text{ max.} - Td$ $Td = 0.060 \sqrt[3]{p^2}$ $Td = 0.060 \sqrt[3]{(0.02777778)^2}$ $Td = 0.060 \sqrt[3]{0.000772}$ $Td = 0.060$ (0.091736) $Td = 0.005504$ $Td = 0.0055$ $d \text{ min.} = 0.2960 - 0.0055$ $d \text{ min.} = 0.2905$	 All thread calculations are to be performed using (p) rounded to 8 decimal places. This figure is rounded to four decimal places to obtain the final value of Td.

Table 4 Example of External Inch Screw Threads, $\frac{19}{64}$ -36 UNS-2A (Cont'd)

Thread Size = $\frac{19}{64}$ -36 or 0.2969 (0.296875 Rounded to Four Decimal Places)-36 UNS-2A		Example of Size Calculation	Additional Information
Characteristic Description			
(3) Maximum external pitch diameter (d_2 max.) = Maximum external major diameter (d max.) – twice the external thread addendum ($2h_{as}$)	d_2 max. = d max. – $2h_{as}$		...
	$2h_{as} = 0.64951905P$		...
	$2h_{as} = 0.64951905 (0.02777778)$		All thread calculations are to be performed using (P) rounded to 8 decimal places.
			...
	$2h_{as} = 0.018042$		...
	d_2 max. = $0.2960 - 0.018042$		...
	d_2 max. = 0.277958		This figure is rounded to four decimal places to obtain the final value of d_2 max.
			...
	d_2 max. = 0.2780		...
	d_2 min. = d_2 max. – Td_2		...
(4) Minimum external pitch diameter (d_2 min.) = Maximum external pitch diameter (d_2 max.) – external pitch diameter tolerance (Td_2)	$Td_2 = 0.0015^3 \sqrt{d_{bsc} + 0.0015 \sqrt{LE} + 0.015^3 \sqrt{P^2}}$		LE (length of engagement) required in this example, is equal to $9P$. See ASME B1.1 for LE applications for other thread series.
			...
	$Td_2 = 0.0015^3 \sqrt{0.2969 + 0.0015 \sqrt{9 (0.02777778)} + 0.015^3 \sqrt{(0.02777778)^2}}$		...
	$Td_2 = 0.0015^3 \sqrt{0.2969 + 0.0015 \sqrt{0.250000} + 0.015^3 \sqrt{0.000772}}$		...
	$Td_2 = (0.0015) (0.667119) + (0.0015) (0.500000) + (0.015) (0.091736)$		...
	$Td_2 = 0.001001 + 0.000750 + 0.001376$		...
	$Td_2 = 0.003127$		This figure is rounded to six decimal places to obtain the final value of the external pitch diameter tolerance Td_2 .
			...
	d_2 min. = $0.2780 - 0.003127$		...
	d_2 min. = 0.274873		This figure is rounded to four decimal places to obtain the final value of d_2 min.
	d_2 min. = 0.2749		...

Table 4 Example of External Inch Screw Threads, $\frac{19}{64}$ -36 UNS-2A (Cont'd)

Thread Size = $\frac{19}{64}$ -36 or 0.2969 (0.296875 Rounded to Four Decimal Places)-36 UNS-2A		
Characteristic Description	Example of Size Calculation	Additional Information
(5) Maximum external UNR minor diameter (d_3 max.) = Maximum external major diameter (d max.) – double height of external UNR thread ($2h_s$)	$d_3 \text{ max.} = d \text{ max.} - 2h_s$ $2h_s = 1.19078493P$ $2h_s = 1.19078493 (0.02777778)$ $2h_s = 0.033077$ $d_3 \text{ max.} = 0.2960 - 0.033077$ $d_3 \text{ max.} = 0.262923$ $d_3 \text{ max.} = 0.2629$	 All thread calculations are to be performed using (P) rounded to eight decimal places. This figure is rounded to four decimal places to obtain the final value of d_3 max. ...
(6) Maximum external UN minor diameter (d_1 max.) = Maximum external major diameter (d max.) – double height of external UN thread ($2h_s$)	$d_1 \text{ max.} = d \text{ max.} - 2h_s$ $2h_s = 1.08253175P$ $2h_s = 1.08253175 (0.02777778)$ $2h_s = 0.030070$ $d_1 \text{ max.} = 0.2960 - 0.030070$ $d_1 \text{ max.} = 0.265930$ $d_1 \text{ max.} = 0.2659$	 For UN threads, $2h_s = 2h_n$ All thread calculations are to be performed using (P) rounded to eight decimal places. This figure is rounded to four decimal places to obtain the final value of d_1 max. ...

GENERAL NOTES:

(a) All dimensions expressed in inches for calculations in this table.

(b) $P = 1/n = 1/36 = 0.02777778$.

Table 5 Example of Internal Inch Screw Threads, $\frac{19}{64}$ -36 UNS-2B

Thread Size = $\frac{19}{64}$ -36 or 0.2969 (0.296875 Rounded to Four Decimal Places)-36 UNS-2B	
Characteristic Description	Example of Size Calculation
(1) Minimum internal minor diameter (D_1 min.) = Basic major diameter (D bsc) – double height of internal thread ($2h_n$)	$D_1 \text{ min.} = D \text{ bsc} - 2h_n$ $2h_n = 1.08253175P$ $2h_n = 1.08253175 (0.02777778)$ $2h_n = 0.030070$ $D_1 \text{ min.} = 0.2969 - 0.030070$ $D_1 \text{ min.} = 0.266830$ $D_1 \text{ min.} = 0.267$
(2) Maximum internal minor diameter (D_1 max.) = Minimum internal minor diameter (D_1 min.) rounded to six decimal places + internal minor diameter tolerance (TD_1)	$D_1 \text{ max.} = D_1 \text{ min. (to six decimal places)} + TD_1$ $TD_1 = 0.25P - 0.4P^2$ $TD_1 = 0.25 (0.02777778) - 0.4 (0.02777778)^2$ $TD_1 = 0.25 (0.02777778) - 0.4 (0.000772)$ $TD_1 = 0.006944 - 0.000309$ $TD_1 = 0.006635$ $TD_1 = 0.0066$ $D_1 \text{ max.} = 0.266830 + 0.0066$ $D_1 \text{ max.} = 0.273465$ $D_1 \text{ max.} = 0.273$

...
 ...
 All thread calculations are to be performed using (P) rounded to 8 decimal places.
 ...

...
 For the Class 2B thread used in this example this figure is rounded to three decimal places to obtain the final value of D_1 min. Other sizes and classes are expressed in a four place decimal. See para. 3.1.3(a)(1).
 ...

See para 3.1.3(a)(2) for exception rule when calculating D_1 max.
 See ASME B1.1 for limitations on use of this formula.
 All thread calculations are to be performed using (P) rounded to 8 decimal places.
 ...

...
 This figure is rounded to four decimal places to obtain the final value of TD_1 .
 ...

...
 For the Class 2B thread used in this example this figure is rounded to three decimal places to obtain the final value of D_1 max. Other sizes and classes are expressed in a four place decimal. See para. 3.1.3(a).
 ...

Table 5 Example of Internal Inch Screw Threads, $19/64$ -36 UNS-2B (Cont'd)

Thread Size = $19/64$ -36 or 0.2969 (0.296875 Rounded to Four Decimal Places)-36 UNS-2B		
Characteristic Description	Example of Size Calculation	Additional Information
(3) Minimum internal pitch diameter (D_2 min.) = Basic major diameter (D bsc) – twice the external thread addendum (h_b)	D_2 min. = D bsc – h_b $h_b = 0.64951905P$ $h_b = 0.64951905 (0.02777778)$ $h_b = 0.018042$ D_2 min. = $0.2969 - 0.018042$ D_2 min. = 0.278858 D_2 min. = 0.2789	 All thread calculations are to be performed using (P) rounded to 8 decimal places. This figure is rounded to four decimal places to obtain the final value of D_2 min. ...
(4) Maximum internal pitch diameter (D_2 max.) = Minimum internal pitch diameter (D_2 min.) + internal pitch diameter tolerance (TD_2)	D_2 max. = D_2 min. + TD_2 $TD_2 = 1.300 (TD_2 \text{ for Class 2A})$ $TD_2 = 1.300 (0.003127)$ $TD_2 = 0.004065$ $TD_2 = 0.0041$ D_2 max. = $0.2789 + 0.0041$ D_2 max. = 0.2830	... The constant 1.300 is for this Class 2B example and will be different for Classes 1B and 3B. See ASME B1.1. For Class 2A pitch diameter tolerance, see Table 4 (4). This figure is rounded to four decimal places to obtain the final value of TD_2.
(5) Minimum internal major diameter (D min.) = Basic major diameter (D bsc)	D min. = D bsc D bsc = 0.296875 D bsc = 0.2969 D min. = 0.2969	... This figure is rounded to four decimal places to obtain the final value of D bsc

GENERAL NOTES:

- (a) All dimensions expressed in inches for calculations in this table.
(b) $P = 1/n = 1/36 = 0.02777778$.

Table 6 Example of External Metric Screw Threads, M12 x 1.75-4g6g

Thread Size = M12 x 1.75-4g6g [Notes (1), (2)]		Example of Size Calculation		Additional Information	
Characteristic Description		Example of Size Calculation		Additional Information	
(1) Maximum external major diameter (d max.) = Basic major diameter (d bsc) – allowance (es)		d max. = d bsc – $ es $; $ es $ is absolute value d bsc = 12.000 $ es = 0.034$ d max. = 12.000 – 0.034 d max. = 11.966 when rounded		 (Tabulated value from ASME B1.13M). This figure is rounded to three decimal places to obtain the final value of d max. ...	
(2) Minimum external major diameter (d min.) = Maximum external major diameter (d max.) – major diameter tolerance (Td)		d min. = d max. – Td d max. = 11.966 $Td = 0.265$ d min. = 11.966 – 0.265 d min. = 11.701 when rounded		... [See Table 6 (1)] (Tabulated value from ASME B1.13M). ... This figure is rounded to three decimal places to obtain the final value of d min.	
(3) Maximum external pitch diameter (d_2 max.) = Basic major diameter (d bsc) – allowance (es) – 0.6495191P or Maximum external major diameter (d max.) – 0.6495191P		d_2 max. = d max. – 0.6495191P d max. = 11.966 $0.6495191P = 0.6495191 (1.75)$ $0.6495191P = 1.1366584$ $0.6495191P = 1.13666$ d_2 max. = 11.966 – 1.13666 d_2 max. = 10.82934 d_2 max. = 10.829 when rounded		... [See Table 6 (1)] ... This figure is rounded to five decimal places which is two places beyond the three decimal place final value d_2 max. This value may also be found in thread data table in ASME B1.13M. ... This figure is rounded to three decimal places to obtain the final value of d_2 max. ...	
(4) Minimum external pitch diameter (d_2 min.) = Maximum external pitch diameter (d_2 max.) – external pitch diameter tolerance (Td_2)		d_2 min. = d_2 max. – Td_2 d_2 max. = 10.829 $Td_2 = 0.095$ d_2 min. = 10.829 – 0.095 = 10.734 d_2 min. = 10.734 when rounded		... [See Table 6 (3)] (Tabulated value from ASME B1.13M). ...	

Table 6 Example of External Metric Screw Threads, M12 x 1.75-4g6g (Cont'd)

Thread Size = M12 x 1.75-4g6g [Notes (1), (2)]			
Characteristic Description	Example of Size Calculation	Additional Information	
(5) Maximum external minor diameter (d_1 max.) (flat form) = Maximum external pitch diameter (d_2 max.) - 0.4330127P	d_1 max. = d_2 max. - 0.4330127P d_2 max. = 10.829 0.4330127P = (0.4330127) (1.75) 0.4330127P = 0.7577722	...	[See Table 6 (3)]
		...	This figure is rounded to five decimal places which is two places beyond the three decimal place final value of d_1 max. This value may also be found in thread data table in ASME B1.13M.
	0.4330127P = 0.75777	...	...
	d_1 max. = 10.829 - 0.75777 d_1 max. = 10.07123	...	This figure is rounded to three decimal places to obtain the final value of d_1 max.
	d_1 max. = 10.071 when rounded	...	...
(6) For reference: Minimum external minor diameter (d_3 min.) (round form) = Minimum external pitch diameter (d_2 min.) - 0.6160254P	d_3 min. = d_2 min. - 0.6160254P d_2 min. = 10.734 0.6160254P = (0.6160254) (1.75) 0.6160254P = 1.0780445	...	[See Table 6 (4)]
		...	This figure is rounded to five decimal places which is two places beyond the three decimal place final value d_3 min. This value may be also be found in thread data table in ASME B1.13M.
	0.6160254P = 1.07804	...	This figure is rounded to three decimal places to obtain the final value of d_3 min.
	d_3 min. = 10.734 - 1.07804 = 9.65596	...	...
	d_3 min. = 9.656 when rounded	...	...

GENERAL NOTE: All dimensions are expressed in millimeters.

NOTES:

- Example of how tabulated values for allowances and tolerances (in accordance with ISO 965-1) are used for the calculation of size limits (in ASME B1.13M tables) for standard diameter/pitch combinations listed in ISO 261.
- For an example of size limit calculations by the use of formulas for sizes not listed see Table 8 (external) and Table 9 (internal).

Table 7 Example of Internal Metric Screw Threads, M12 x 1.75-6H

Thread Size = M12 x 1.75-6H [Notes (1), (2)]		Example of Size Calculation	Additional Information
Characteristic Description			
(1) Minimum internal major diameter ($D_{\min.}$) = Basic internal major diameter (D_{bsc}) + allowance (E)	$D_{\min.} = D_{\text{bsc}} + E $; $ E $ is absolute value	...	[See Table 6 (1)]
	$D_{\text{bsc}} = d_{\text{bsc}} = 12.000$	...	Tabulated value from ASME B1.13M.
	$ E_H = 0$	...	...
	$D_{\min.} = 12.000 + 0.000$	...	...
(2) Minimum internal pitch diameter ($D_{2\min.}$) = Basic internal major diameter (D_{bsc}) - 0.6495191P + allowance (E) or Minimum internal major diameter ($D_{\min.}$) - 0.6495191P	$D_{2\min.} = D_{\min.} - 0.6495191P$	...	[See Table 7 (1)]
	$D_{\min.} = 12.000$	...	...
	$0.6495191P = 0.6495191 (1.75)$	...	This figure is rounded to five decimal places which is two places beyond the three decimal place final value of $D_{2\min.}$.
	$0.6495191P = 1.1366584$	...	This value may also be found in thread data table in ASME B1.13M.
	$0.6495191P = 1.13666$	...	...
	$D_{2\min.} = 12.000 - 1.13666$	...	This figure is rounded to obtain the final value of minimum pitch diameter.
	$D_{2\min.} = 10.86334$	...	...
	$D_{2\min.} = 10.863$ when rounded	...	...
	$D_{2\max.} = D_{2\min.} + TD_2$	...	[See Table 7 (2)]
	$D_{2\min.} = 10.863$	...	Tabulated value from ASME B1.13M for internal tolerance grade 6.
(3) Maximum internal pitch diameter ($D_{2\max.}$) = Minimum internal pitch diameter ($D_{2\min.}$) + internal pitch diameter tolerance (TD_2)	$TD_2 (6) = 0.200$	...	...
	$D_{2\max.} = 10.863 + 0.200$	...	...
	$D_{2\max.} = 11.063$	...	...
	$D_{1\min.} = D_{\min.} - 1.0825318P$	...	[See Table 7 (1)]
	$D_{\min.} = 12.000$	...	...
	$1.0825318P = (1.0825318) (1.75)$	...	This figure is rounded to five decimal places which is two places beyond the three decimal place final value $D_{1\min.}$.
	$1.0825318P = 1.8944306$	...	This value may also be found in thread data table in ASME B1.13M.
	$1.0825318P = 1.89443$	...	...
	$D_{1\min.} = 12.000 - 1.89443$	...	This figure is rounded to three decimal places to obtain the final value of minimum pitch diameter.
	$D_{1\min.} = 10.10557$	...	...
(4) Minimum internal minor diameter ($D_{1\min.}$) = Minimum internal major diameter ($D_{\min.}$) - 1.0825318P	$D_{1\min.} = 10.106$ when rounded	...	...

Table 7 Example of Internal Metric Screw Threads, M12 x 1.75-6H (Cont'd)

Thread Size = M12 x 1.75-6H [Notes (1), (2)]		
Characteristic Description	Example of Size Calculation	Additional Information
(5) Maximum internal minor diameter ($D_{1\max}$) = Minimum internal minor diameter ($D_{1\min}$) + internal minor diameter tolerance (TD_1)	$D_{1\max} = D_{1\min} + TD_1$ $D_{1\min} = 10.106$ $TD_1(6) = 0.335$ $D_{1\max} = 10.106 + 0.335$ $D_{1\max} = 10.441$	... [See Table 7 (4)] Tabulated value from ASME B1.13M for internal tolerance grade 6.
(6) For reference: Maximum internal major diameter ($D_{\max}$) = Maximum internal pitch diameter ($D_{2\max}$) + $0.7938566P$	$D_{\max} = D_{2\max} + 0.7938566P$ $D_{2\max} = 11.063$ $0.7938566P = (0.7938566) 1.75$ $0.7938566P = 1.3892490$ $0.7938566P = 1.38925$ $D_{\max} = 11.063 + 1.38925$ $D_{\max} = 12.45225$ $D_{\max} = 12.452$ when rounded	... [See Table 7 (3)] ... This figure is rounded to five decimal places which is two places beyond the three decimal place final value $D_{\max}$. This value may also be found in thread data table in ASME B1.13M. ... This figure is rounded to three decimal places to obtain the final value of maximum major diameter. ...

GENERAL NOTE: All dimensions are expressed in millimeters.

NOTES:

- (1) Example of how tabulated values for allowances and tolerances (in accordance with ISO 965-1) are used for the calculation of size limits (in ASME B1.13M tables) for standard diameter/pitch combinations listed in ISO 261.
- (2) For an example of size limit calculations by the use of formulas for sizes not listed see Table 9 (external) and Table 10 (internal).

Table 8 Example of External Metric Screw Threads, M13 x 0.9-4g6g

Thread Size = M13 x 0.9-4g6g [Notes (1), (2)]		
Characteristic Description	Example of Size Calculation	Additional Information
(1) Maximum external major diameter ($d_{\max}$) = Basic major diameter (d_{bsc}) – allowance (es)	$d_{\max} = d_{\text{bsc}} - es $; $ es $ is absolute value $d_{\text{bsc}} = 13.000$ $ es_g = 0.015 + 0.011P$ $ es_g = 0.015 + 0.011(0.9)$ $ es_g = 0.0249$ $ es_g = 0.025$ when rounded $d_{\max} = 13.000 - 0.025$ $d_{\max} = 12.975$	... This is the final value of the basic major diameter and is therefore expressed with three decimal places. es for tolerance position g is not tabulated for $P = 0.9$ so it must be calculated. ... This figure is rounded to three decimal places to obtain the final absolute value of allowance for tolerance position g.
(2) Minimum external major diameter ($d_{\min}$) = Maximum external major diameter ($d_{\max}$) – tolerance (Td)	$d_{\min} = d_{\max} - Td$ $d_{\max} = 12.975$ $Td(6) = 0.18 \sqrt[3]{\frac{P^2}{\sqrt{P}}} - \frac{0.00315}{\sqrt{P}}$ $Td(6) = 0.18 \sqrt[3]{\frac{0.9^2}{\sqrt{0.9}}} - \frac{0.00315}{\sqrt{0.9}}$ $Td(6) = 0.18 \sqrt[3]{\frac{0.81000}{0.94868}} - \frac{0.00315}{0.94868}$ $Td(6) = 0.18(0.93217) - 0.00315$ $Td(6) = 0.16779 - 0.00332$ $Td(6) = 0.16447$ $Td(6) = 0.164$ when rounded $d_{\min} = 12.975 - 0.164$ $d_{\min} = 12.811$	... [See Table 8 (1)] Td for tolerance grade 6 is not tabulated for $P=0.9$ so it must be calculated. This figure is rounded to three decimal places to obtain the final value of major diameter tolerance for tolerance grade 6.
(3) Maximum external pitch diameter ($d_2_{\max}$) = [Basic major diameter (d_{bsc}) – allowance (es)] – 0.6495191P or Maximum external major diameter ($d_{\max}$) – 0.6495191P	$d_2_{\max} = d_{\max} - 0.6495191P$ $d_{\max} = 12.975$ $0.6495191P = 0.6495191(0.9)$ $0.6495191P = 0.5845672$ $0.6495191P = 0.58457$ $d_2_{\max} = 12.975 - 0.58457$ $d_2_{\max} = 12.39043$ $d_2_{\max} = 12.390$ when rounded	... [See Table 8 (1)] ... This figure is rounded to five decimal places to obtain the final value. This figure is rounded to obtain the final value of $d_2_{\max}$. ...

Table 8 Example of External Metric Screw Threads, M13 x 0.9-4g6g (Cont'd)

Thread Size = M13 x 0.9-4g6g [Notes (1), (2)]		
Characteristic Description	Example of Size Calculation	Additional Information
(4) Minimum external pitch diameter (d_{2min}) = Maximum external pitch diameter (d_{2max}) – external pitch diameter tolerance (Td_2)	$d_{2min} = d_{2max} - Td_2$ $d_{2max} = 12.390$ $Td_2 (4) = 0.63Td_2 (6)$ $Td_2 (4) = 0.63 [0.09P^{0.4}d^{0.1}]$ $Td_2 (4) = 0.63 [0.09 (0.9)^{0.4} (13)^{0.1}]$ $Td_2 (4) = 0.63 [0.09 (0.95873) (1.29239)]$ $Td_2 (4) = 0.63 [(0.09) (1.23905)]$ $Td_2 (4) = 0.63 (0.11151)$ $Td_2 (4) = 0.07025$ $Td_2 (4) = 0.070$, when rounded $d_{2min} = 12.390 - 0.070$ $d_{2min} = 12.320$	... [See Table 8 (3)] Td_2 for tolerance grade 4 is not tabulated for $P=0.9$ so it must be calculated. This figure is rounded to three decimal places to obtain the final value of pitch diameter tolerance for tolerance grade 4.
(5) Maximum external minor diameter (d_{1max}) (flat form) = Maximum external pitch diameter (d_{2max}) – 0.4330127P	$d_{1max} = d_{2max} - 0.4330127P$ $d_{2max} = 12.390$ $0.4330127P = 0.4330127 (0.9)$ $0.4330127P = 0.3897114$ $0.4330127P = 0.38971$ $d_{1max} = 12.390 - 0.38971$ $d_{1max} = 12.00029$ $d_{1max} = 12.000$, when rounded	... [See Table 8 (3)] ... This figure is rounded to five decimal places, which is two decimal places beyond the three decimal place value of d_{1max}. This figure is rounded to three decimal places to obtain the final value of maximum minor diameter. ...
(6) For reference: Minimum external minor diameter (d_{3min}) (round form) = Minimum external pitch diameter (d_{2min}) – 0.6160254P	$d_{3min} = d_{2min} - 0.6160254P$ $d_{2min} = 12.320$ $0.6160254P = 0.6160254 (0.9)$ $0.6160254P = 0.5544229$ $0.6160254P = 0.55442$ $d_{3min} = 12.320 - 0.55442$ $d_{3min} = 11.76558$ $d_{3min} = 11.766$ when rounded	... [See Table 8 (4)] ... This figure is rounded to five decimal places, which is two decimal places beyond the three decimal place final value of d_{3min}. This figure is rounded to obtain the final value of minimum minor diameter. ...

GENERAL NOTE: All dimensions are expressed in millimeters.

NOTES:

- (1) For an example of how tabulated values for allowances and tolerances (in accordance with ISO 965-1) are used for the calculation of size limits (in ASME B1.13M tables) for standard diameter/pitch combinations listed in ISO 261, see Table 6 and 7.
- (2) Example of size limit calculations by the use of formulas for diameter/pitch combinations not listed in ISO 261.

Table 9 Example of Internal Metric Screw Threads, M13 x 0.9-6H

Thread Size = M13 x 0.9-6H [Notes (1), (2)]		
Characteristic Description	Example of Size Calculation	Additional Information
(1) Minimum internal major diameter ($D_{1\min.}$) = Basic internal major diameter (D_{bsc}) + allowance (EI)	$D_{\min.} = D_{\text{bsc}} + EI $; $ EI $ is absolute value $D_{\text{bsc}} = d_{\text{bsc}} = 13.000$ $EI_H = 0$ $D_{\min.} = 13.000 + 0.000$ $D_{\min.} = 13.000$	... [see Table 8 (1)] Tabulated value from ASME B1.13M for internal tolerance position H
(2) Minimum internal pitch diameter ($D_{2\min.}$) = Basic internal major diameter ($D_{\min.}$) - 0.6495191P + allowance (EI) or Minimum internal major diameter ($D_{\min.}$) - 0.6495191P	$D_{2\min.} = D_{\min.} - 0.6495191P$ $D_{\min.} = 13.000$ $0.6495191P = 0.6495191 (0.9)$ $0.6495191P = 0.5845672$ $0.6495191P = 0.58457$ $D_{2\min.} = 13.000 - 0.58457$ $D_{2\min.} = 12.41543$ $D_{2\min.} = 12.415$ when rounded	... [See Table 9 (1)] ... This figure is rounded to five decimal places, which is two places beyond the three decimal place final value of $D_{2\min.}$ This figure is rounded to obtain the final value of minimum pitch diameter.
(3) Maximum internal pitch diameter ($D_{2\max.}$) = Minimum internal pitch diameter ($D_{2\min.}$) + internal pitch diameter tolerance (TD_2)	$D_{2\max.} = D_{2\min.} + TD_2$ $D_{2\min.} = 12.415$ $TD_2 (6) = 1.32TD_2 (6)$ $TD_2 (6) = 1.32 [0.09P^{0.4} d^{0.1}]$ $TD_2 (6) = 1.32 [0.09 (0.9)^{0.4} (13)^{0.1}]$ $TD_2 (6) = 1.32 [0.09 (0.95873) (1.29239)]$ $TD_2 (6) = 0.14720$ $TD_2 (6) = 0.147$, when rounded $D_{2\max.} = 12.415 + 0.147$ $D_{2\max.} = 12.562$	... [See Table 9 (2)] TD_2 (or TD_2) for tolerance grade 6 is not tabulated for $P = 0.9$ so it must be calculated. This figure is rounded to three decimal places to obtain the final value of pitch diameter tolerance for tolerance grade 6.
(4) Minimum internal minor diameter ($D_{1\min.}$) = Minimum internal major diameter ($D_{\min.}$) - 1.0825318P	$D_{1\min.} = D_{\min.} - 1.0825318P$ $D_{\min.} = 13.000$ $1.0825318P = 1.0825318 (0.9)$ $1.0825318P = 0.9742786$ $1.0825318P = 0.97428$ $D_{1\min.} = 13.000 - 0.97428$ $D_{1\min.} = 12.02572$ $D_{1\min.} = 12.026$ when rounded	... [See para. 3.2.8 (1)] ... This figure is rounded to five decimal places, which is two places beyond the three decimal place final value of $D_{1\min.}$ This figure is rounded to three decimal places to obtain the final value of minimum pitch diameter.